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The Art of Bond Graph Construction for Transducers

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13. ABSTRACT (Maximum 200 words)

The mathematical modeling of transducer by bond graphs possesses notable advantages. While these advantages come to true light in very complex devices, the method of bond graphs is applied here to elementary mechanical-acoustical-electrical devices, thus serving as an introduction to bond graph modeling procedures. The devices selected for illustration are (1) the longitudinal vibrator, (2) the electrodynamic microphone, (3) the moving armature transducer. In addition to these examples, a preview of 4-pole theory is presented to allow easy comparisons between bond graph modeling and conventional methods.

These notes are intended to accompany another publication entitled, "Monograph—Bond Graph Modeling of the Dynamics of Physical Systems," S. Hanish, Naval Research Laboratory, Washington, DC.

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CONTENTS

THE ART OF
BOND GRAPH CONSTRUCTION FOR TRANSDUCERS

<u>Bulletin #1 - Longitudinal Vibrator</u> (A 2-Port Model)
(Ref.: F. Hunt "Electroacoustics" and S. Hanish "Treatise on Acoustic Radiation" Vol. II NRL)

The longitudinal vibrator can conveniently serve as a prototype in the art of bond graph construction.

Suppose first we have a moderately long thin elastic bar of constant cross sectional area and assume it is in steady state vibration at a fixed frequency. We propose to construct a bond graph model of this bar. Our procedure is outlined in a sequence of steps.

(1) As an initial step we perform a <u>dynamic analysis</u> of the motion of the bar. But immediately we note a different approach: in conventional analysis we formulate an equation of velocity resulting from applied force; in a different approach, because a bond graph pictures a flow of <u>power</u>, we seek to find the velocity at every point in the bar, as well as the force at every point, in terms of the force and velocity at one end of the bar. Let us set aside the physical causes of this force and velocity at the input point (that is, bar end). Because the bar is an elastic continuum the force and velocity at the second end are easily obtained. Analysis therefore leads to two equations relating forces F_1, F_2 and velocities V_1, V_2.
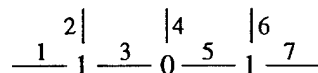
$$F_2 = F_2(F_1, V_1)$$
$$V_2 = V_2(F_1, V_1)$$

In most common applications' we find the functional relations to be linear, and the power variables decoupled, i.e.
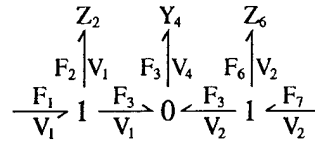
$$F_2 = AF_1 + BV_1$$
$$V_2 = CF_1 + DV_1$$

Here A,B,C,D apply only to the terminii of the bar. By so doing we have converted the bar description from that of a continuum to a system of 2 degrees-of-freedom, namely the velocities of the ends of the bar. On dimensional considerations it is seen that A,B,C,D are impedances, or admittances, or ratios of impedance's.

The above set of equations in F_2, V_2, obtained by dynamic analysis, has no relation to any preconceived model, be it electrical analogies or bond graphs, or signal graphs, or chose what one will.

(2) The second step in the procedure introduces modeling by bond graphs. To begin, one identifies each one of two degrees of-freedom with a "one-junction" (written as, -1-), and because these are coupled, the coupling (by bond graph rules) is a "zero-junction" (written as -0-). The primitive graph is then,

$$\underset{1}{\overset{2|}{\rule{0pt}{0pt}}} 1 \overset{3}{\rule{1cm}{0.4pt}} 0 \overset{5}{\rule{1cm}{0.4pt}} 1 \overset{7}{\rule{1cm}{0.4pt}}$$

with vertical bonds 2, 4, 6

The digits identify the power bonds, which are paths of power flow. Bonds #1, #7 are "inputs," that is "ports." Bonds #2, #4, #6 are reactive impedances or admittances associated with a lumped parameter 2 degree-of-freedom system. Bonds #3, #5 are interior coupling bonds. Completing a more elaborate graph, one has,

$$\begin{array}{ccccc}
Z_2 & & Y_4 & & Z_6 \\
\uparrow & & \uparrow & & \uparrow \\
F_2 \Big| V_1 & & F_3 \Big| V_4 & & F_6 \Big| V_2 \\
\dfrac{F_1}{V_1} \nearrow 1 & \dfrac{F_3}{V_1} \rightarrow 0 & \leftarrow & \dfrac{F_3}{V_2} 1 & \leftarrow \dfrac{F_7}{V_2}
\end{array}$$

The arrow directions are arbitrary, but it is advantageous to assign them "all right pointing," "all left pointing," or "symmetrical," (as above), Z_2, Z_6 are mechanical impendances, and Y4 is a mechanical admittance.

The above graph now permits writing of "graph equations" in accordance with the bond graph rules (1) that 1-junctions represent the operation of force summation, and (2) the 0-junctions represent the operation of velocity summation. The "graph equations" are then:

(a) $F_1 = Z_2 V_1 + F_3$; (b) $V_1 = Y_4 F_{12} - V_2$; (c) $F_7 = Z_6 V_2 + F_3$

Here the coupling force F_{12} is the same as as F_3.

By construction, the model has now three unknowns, namely Z_2, Y_4, Z_6.

(3) The third, and crucial step in the construction of the bond graph model is to rearrange the "graph equations" so that they appear as the set,

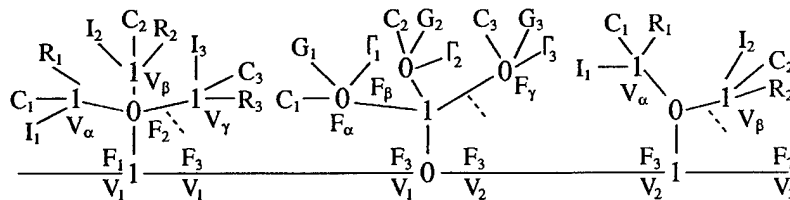$$F_7 = G F_1 + H V_1; \quad V_2 = J F_1 + K V_1$$

in which G, H, J, K are algebraic functions of Z_2, Y_4, Z_6, and are therefore unknown. To find them explicitly we return to the dynamic equations obtained by analysis, and equate comparable coefficients:

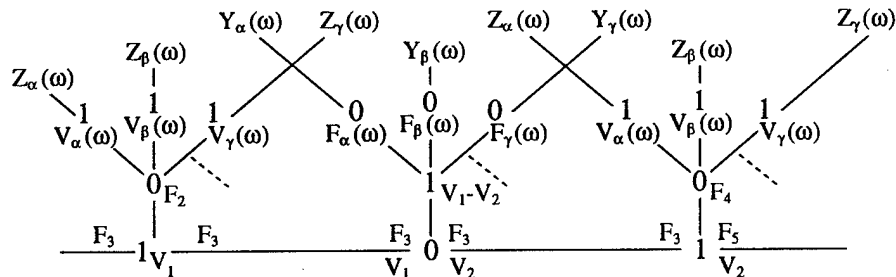(a) $G = A$; (b) $H = B$; (c) $J = C$; (d) $K = D$

Because A, B, C, D are known, this set allows Z_2, Y_4, Z_6 to be known function of one, two, three or all four of them.

The model, as a first approximation (i.e. 2 degrees-of-freedom) is now completed. It is seen to be a "one-zero-one" structure, that is, a conventional "Tee" in electric circuit theory which, in itself, is an approximation to a short transmission line. However it is to be emphasized that the bond graph model is independent of electric circuit, or electric transmission line theory.

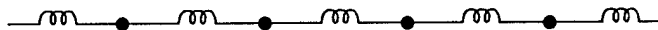(4) The fourth step in the procedure is to introduce modal description of vibration of the bar by expanding Z2, Y4, Z6 in resonant or anti-resonant modes. This is possible because Z_2, Y_4, Z_6 are transcendental functions of frequency. The type of expansion is optional. We choose here to expand Z_2, Z_6 in resonant modes, and Y_4 in antiresonant modes. The bond graph symbols for expansions are "fans of bonds."

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Here v_α, v_β, v_γ ... are modal velocities; F_α, F_β, F_γ ... are modal forces; C_α, C_β, C_γ ... are modal compliances; R_α, R_β, R_γ ... are modal resistances. Similarly, $\mathcal{C}$, G, Γ are inverses of mass, resistance and compliance. All modal parameters (I, C, R, $\mathcal{C}$, G, Γ) are known from the results of step #3 above.

The bond graph of the mechanical motion of the bar is now completed.

Explanation

(1) the lumped parameter model is a cascade of masses and springs.

(2) V_1 is coupled to V_2 via a cascade of model masses and modal springs, the magnitudes of which are frequency dependent.

(3) at any frequency, velocity V_1 is a sum of modal velocities $V_\alpha(\omega)$, $V_\beta(\omega)$, $V_\gamma(\omega)$, etc. which are determined by modal $Z_\alpha(\omega)$, $Z_\beta(\omega)$, etc.

(4) at modal resonant frequencies ω_1^R, ω_2^R, ... etc., the $Z_\alpha(\omega_1^R)$, $Z_\beta(\omega_2^R)$, etc. are minimized so that V_α or V_β ... etc. is maximized.

(5) at any frequency ω the force $F_3(\omega)$ which couples V_1 to V_2 is a sum of modal forces $F_\alpha(\omega)$, $F_\beta(\omega)$, $F_\gamma(\omega)$, etc.

(6) at modal antiresonant frequencies ω_1^A, ω_2^A, etc., the $Y_\alpha(\omega_1^A)$, $Y_\beta(\omega_2^A)$, etc., are minimized so that $F_\alpha(\omega_1^A)$, $F_\beta(\omega_2^A)$, etc., are maximized.

(7) the assignment of directions of power flow at any frequency ω is purely arbitrary, but consistent choices can be made.

(8) the equations of motion can be written directly by summing forces at 1-junctions and/or summary velocities at 0-junctions.

Observations

(a) If the bar is driven by an external source of power the model then contains a source bond. Inspection of the physical requirements shows at what point in the graph it is to be attached. For example, if the bar itself is piezoelectric the driving force in a 2 degree-of-freedom model shares the same velocity difference as the admittance Y_4. Its representation is therefore,

(b) Since the bar is an elastic continuum more points of velocity can be used in the dynamic analysis as needed. For example, if a compliance load is attached, x units of distance from the end its velocity V_3 can be represented (in approximation) as the third degree-of-freedom in a 3 degree-of-freedom system, i.e.
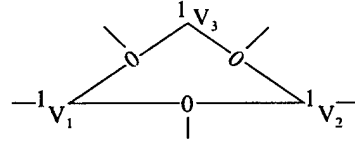

(c) Since an elastic continuum may vibrate in several different modes—longitudinal, shear, twist, bulge, radial etc.—it is a common occurrence of a vibratory motion that two or more of these modes exist together over particular frequency ranges. Cross coupling between them is then to be expected. For example, suppose a vibration is simultaneously longitudinal, shear, and radial and assume the modeling makes the following assignments:

longitudinal mode: 2-d.o.f. $(V\ell_1, V\ell_2)$
shear mode: 2-d.o.f. (V_{S_1}, V_{S_2})
radial mode: 1-d.o.f. (V_R)

(Note: d.o.f. means 'degree of freedom')

Assume further that analysis shows cross couplings of compliance, inertance, and resistance between respective velocities. An illustrative graph may have the following appearance:
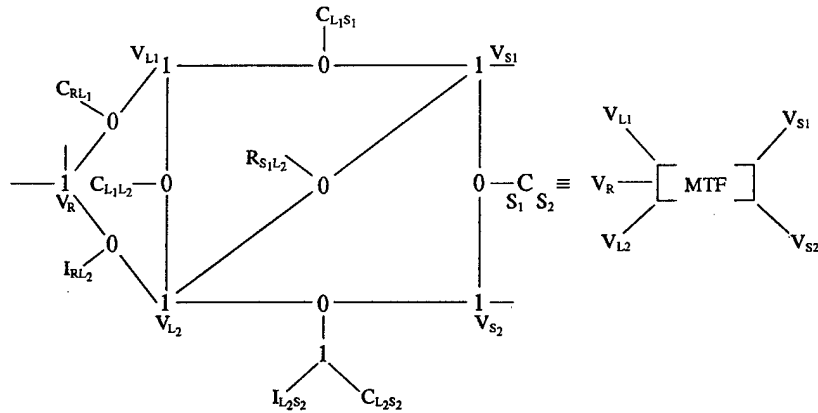
MTF ≡	V_R	V_{L_1}	V_{L_2}	V_{S_1}	V_{S_2}
V_R		C_{RL_1}	I_{RL_2}		
V_{L_1}	C_{L_1R}		$C_{L_2L_1}$	$C_{L_1S_1}$	
V_{L_2}	I_{L_2R}	$C_{L_1L_2}$		$R_{L_2S_1}$	$I_{L_2S_2}, C_{L_2S_2}$
V_{S_1}		$C_{S_1L_1}$	$R_{S_1L_2}$		$C_{S_1S_2}$
V_{S_2}			$I_{S_2L_2}, C_{S_2L_2}$	$C_{S_2S_1}$	

Cross couplings modes are therefore easily made visible and tractable
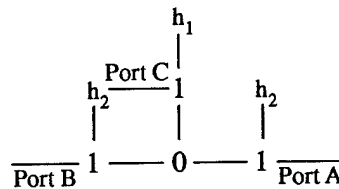
4

An Alternative Approach

A 3-Port Model of a Longitudinal Vibrator

(1) As a first step we anticipate a model of the bar to have the character of a multiport representation. A port by definition is a location on the surface of the bar where power may enter or exit during motion of the bar. In bond graph theiry a port is pictured as a short straight line terminating at one end in a junction of the bond graph and directed at the other end away from the graph.

Inspection of the bar shows the existence of several possible ports. We chose three: one at each end of the bar, and one on the flat surface $l_b l_b$. Each port is identified by its power variables of effort and flow properly subscripted.
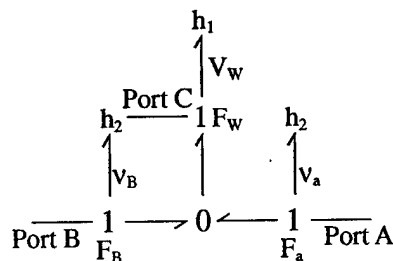
(2) A second step is to perform a dynamic analysis of the motion of the bar for the condition of steady state. The objective of this analysis is to associate with each port an explicit mathematical form of its input impedance or admittance. This procedure leads to a multiport solution of the motion of the bar.

(3) The next step is to construct an elementary bondgraph of the vibrating bar. Each port (in the form of a short straight line) is attached to a 1-junction, that is to an "effort divider." Since the motion at the ends are coupled to each other, their ports (here designated as A, B) are connected via a 0-junction, that is, to an "effort divider." The flow of the third port, here designated as port C, is coupled to ports A, B only through the already established 0-junction. The primitive bond graph can now readily be constructed:

$$
\begin{array}{c}
h_1 \\
| \\
\overset{\text{Port C}}{h_2 \text{———} 1} \qquad h_2 \\
| \qquad\quad | \qquad\quad | \\
\overline{\text{Port B}}\ 1 \text{——} 0 \text{——} 1\ \overline{\text{Port A}}
\end{array}
$$

Here, the symbol h_2 is the input self admittance of port B, and of port A. The symbol h_1 is the input self admittance of port C.

(4) Since the bar in question is mechanical, its power variables are force F and velocity V. For various reasons the makers of models have found it advantageous to choose F to be "flow," and V to be "effort." A more complete bond graph incorporates these choices:

$$
\begin{array}{c}
h_1 \\
\uparrow V_W \\
\overset{\text{Port C}}{h_2 \text{———} 1}\, F_W \qquad h_2 \\
\uparrow V_B \qquad \uparrow \qquad \uparrow V_a \\
\overline{\text{Port B}}\ \underset{F_B}{1} \text{——}\!\rightarrow 0 \leftarrow\!\text{——} \underset{F_a}{1}\ \overline{\text{Port A}}
\end{array}
$$

(Note that arrow directions are arbitrary, but conform to convention.)

(5) We next formulate the <u>constitutive equations</u> relating force to velocity at each port. For the choice of effort and flow variables made earlier these are

$$(1)\ v_a = h_2 F_a;\ (2)\ v_b = h_2 F_b;\ (3)\ v_w = h_1 F_w$$

It is seen that h_1, h_2 are <u>mechanical admittances.</u>

(6) Explicit forms of h_1, h_2 are available in the literature (Ref. 1). Our next step is to incorporate them in the constitutive equations

$$(1)\ v_a = \frac{h_F F_a}{j\, \tan\left(\frac{w\ell}{c}\right)}\quad (2)\ v_b = \frac{h_F F_b}{j\, \tan\left(\frac{w\ell}{c}\right)}\quad (3)\ v_w = j\, h_F \sin\left(\frac{w\ell}{c}\right)$$

$$h_F = \frac{1}{\ell_b \ell_d} \times \frac{1}{\rho c};\ c^2 = \frac{E}{\rho}$$

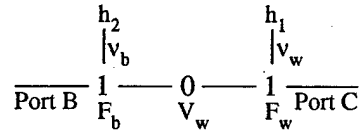$$E = \text{modulus of electricity}\left(\text{units: } \frac{N}{m^e}\right)$$

$$\rho = \text{mass of the bar per unit volume}\left(\text{units: } N\,\text{sec}^2/m^4\right)$$

(7) The bond graph model is now completed. For the choices of power variables it is seen to have the form of a "tee" (i.e. — 1 — 0 — 1 —)
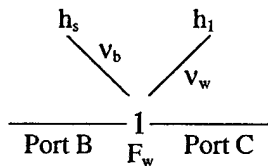
(Ref. W.P. Mason, "Physical Acoustics," Academic Press, New York, 1964.)

<u>Example</u> (a 3-port model of a quarter-wave resonator).

If velocity v_a is made zero by fixing end A of the bar in a wall, the graph constructed above takes on the form,

$$
\begin{array}{ccccc}
 & h_2 & & h_1 & \\
 & |v_b & & |v_w & \\
\overline{\text{Port B}} & 1\!\!-\!\!-\!\! & 0 \!\!-\!\!-\!\! & 1 & \overline{\text{Port C}} \\
 & F_b & V_w & F_w &
\end{array}
$$

Since the 0-junction now has no exterior bond one sets $F_b = F_w$. The reduced graph is now,

$$
\begin{array}{ccc}
 h_s & & h_1 \\
 \diagdown v_b & & \diagup v_w \\
 & 1 & \\
 \overline{\text{Port B}} & F_w & \overline{\text{Port C}}
\end{array}
$$

The admittances h_1, h_2 are now in series, each having the flow F_w. They can be added to form h,

$$h = h_1 + h_2 = j\, h_F \sin\left(\frac{w\ell}{c}\right) + \frac{1}{j\, h_F \tan\left(\frac{w\ell}{c}\right)} = j\, h_F \tan\left(\frac{w\ell}{c}\right)$$

Setting $v = v_b + v_w$ (note that these v's are 'across' variables, that is 'efforts.') the graph is simplified to,

$$
\begin{array}{c}
h \\
\mid v \qquad\qquad v = h\,F_w \\
\overline{\text{Port B}} \;\; \dfrac{1}{F_w} \;\; \overline{\text{Port C}}
\end{array}
$$

Since h is a transcendental function it can be expanded in modes. We choose antiresonant modes, namely modes wherein at any as a sequence of antiresonant frequencies the 'across' variable (here velocity v) is maximized and the 'through' variable (F_w) is minimized. The graph then becomes:

$=$ a quarter-wave longitudinal resonator
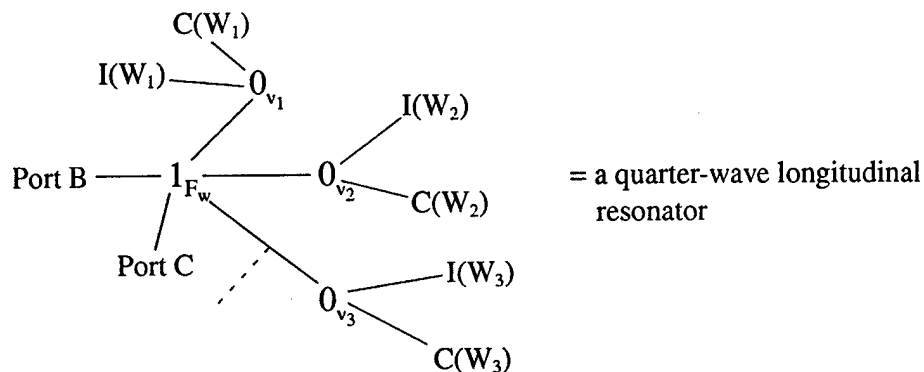
Other examples may be found in the cited references.

Tauchspulenmikrophone

Bulletin #2 - Electrodynamic Microphone (Cascade Model)

In the book "Grundlagen Der Sechnischen Akustik" by W. Reichardt,* there appears on page 305 a description of a Tauchspulenmikrophone (plunger spool microphone = electrodynamic microphone). Its essential parts are reproduced below.
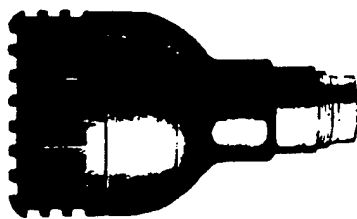
Electrodynamic microphone
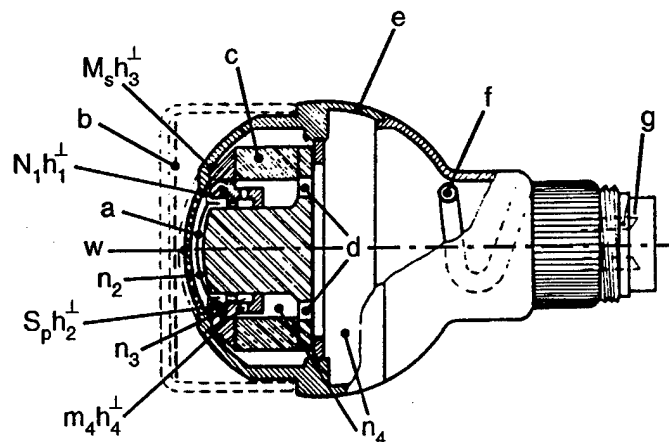
Cross section of electrodynamic microphone

Here n, m, h are lumped compliance, mass, and resistance respectively.

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*Akademische Verlagsgesellshaf Geest Portig K. - G. Leipzig 1968

Acoustic Description

This device is an <u>acoustic pressure</u> receiver. Sound pressure incident from the left passes through screen b and protective grid w to reach the flexible diaphragm a to which a spool of wire is attached. Directly behind the diaphragm are a series of cavities and orifices which perform the function of smoothing the frequency response. The spool of wire is transverse to a static (d.c.) magnetic field created by permanent magnet c. In the rear cavity is an equalizer tube whose orifice f admits "atmospheric pressure" via tube feed g.

Bond Graph Model

A. Construction of a bond graph model of this device requires an understanding of the acoustic performance of cavities and orifices. To illustrate this performance we propose to make a bondgraph model of the following simple cavity and orifice structure.
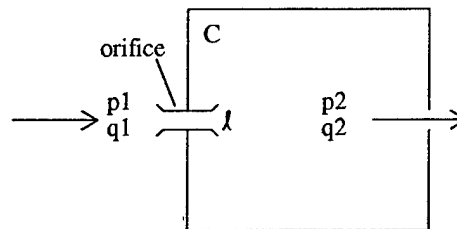
We note:

(1) the flow q is hydrodynamic compressible flow, $q_1 \neq q_2$
(2) the orifice length l is "short", meaning it has constant flow q_1
(3) all variables p_1, p_2, q_1, q_2 are sinusoidal steady state.
(4) a flow resistance R is assumed to be present in the orifice.
(5) all pressures are relative to atmospheric pressure.

The corresponding bond graph is easily constructed:
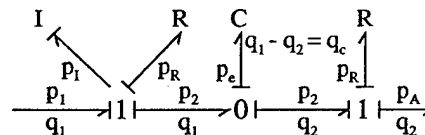
Explanation

(a) the 1-junction (velocity q_1) is a "pressure divider", i.e. the input pressure p_1 is "divided" by the orifice: part p_1 accelerates an air mass m (units Ns^2/m^5), and part p_R (Ns^2/m^5) overcomes the orifice resistance. After subtraction of p_1 and p_R the pressure p_2 in the chamber remains.

(b) the 0-junction is a "volume flow divider" i.e. the input flow q_1 in chamber #2 is partly diverted in amount q_1-q_2 to be associated with the compression/expansion of the chamber volume acting as a "hydrodynamic spring" of compliance C, leaving exit flow q_2 as remainder. The compliance C equals 1/K where K (N/m^5) is the "spring constant."

(c) the 1-junction (velocity q_2) divides p_2 into p_R and p_A.

(d) the directions of power flow through the <u>junctions</u> reverses in the second half of the sinusoi-

dal cycle, but not the directions of power flow in load elements I, R, C.

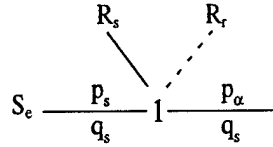(e) the assignment of causality allows the formation of the constitutive equations for I, R, C:

$$(1)\ q_I = \frac{1}{I}\!\int p_I dt \qquad (2)\ p_R = R q_R \qquad (3)\ p_c = \frac{1}{C}\!\int q_c dt$$

B. We now proceed with the construction of the bond graph model. The component parts are,

- (a) (forward) source
- (b) diaphragm
- (c) (smoothing) orifices and cavities
- (d) equalizing tube
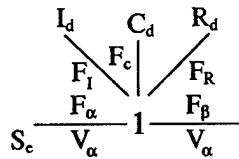- (e) (rear) source

(a) (forward) source

Since the microphone is a pressure sensor we designate the exterior driver to be an "effort source" Se. Because of the presence of screen and grill we take the source impedance to be the acoustic resistance R_s. To construct the bond graph of this source, the rule is invoked that an effort source has the operational significance of a 0-junction, and therefore must be followed by a 1-junction. The model is therefore,
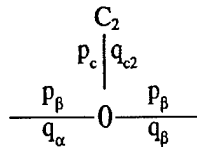
$$S_e \ \underset{q_s}{\overset{p_s}{\rule{2cm}{0.4pt}}} \ 1 \ \underset{q_s}{\overset{p_\alpha}{\rule{2cm}{0.4pt}}}$$

with R_s and R_r bonds into the 1-junction.

R_r has been added to account for radiation resistance, if not negligible.
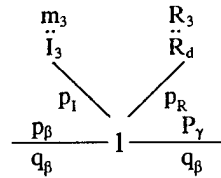
(b) diaphragm

The diaphragm is a mechanical component characterized by the power variables force F; velocity V_α. It is assumed in the first approximation to have rigid piston motion, featuring inertance Id, capacitance C_α, a resistance R_d. In operation the pressure difference between the opposing sides of this piston excite it into motion. The bond graph model is therefore an "effort divider," that is , a 1-junction:

$$S_e \ \underset{V_\alpha}{\overset{F_\alpha}{\rule{2cm}{0.4pt}}} \ 1 \ \underset{V_\alpha}{\overset{F_\beta}{\rule{2cm}{0.4pt}}}$$

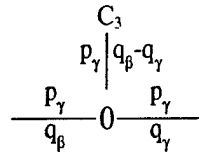with I_d (F_I), C_d (F_c), R_d (F_R) bonds into the 1-junction.

(c) (smoothing) orifices and cavities. For identification of chambers see subscripts on cross-section. The first cavity behind the diaphragm is designated by the symbol C_2 (compliance #2). Volume velocity q_α enters and velocity q_β exits. The bond graph model is therefore a "velocity divider," i.e. a 0-junction,

$$\underset{q_\alpha}{\overset{p_\beta}{\rule{2cm}{0.4pt}}} \ 0 \ \underset{q_\beta}{\overset{p_\beta}{\rule{2cm}{0.4pt}}}$$

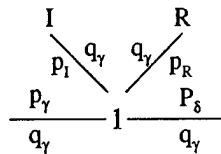with C_2 (p_c, q_{c2}) bond into the 0-junction.

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(d) the first orifice connects chamber #2 to chamber #3. Its volume velocity is q_β. Pressure differences between the two sides of the orifice (tube) accelerate a mass m_3 and overcome a resistance R_3. The bond graph model is therefore an "effort divider" i.e. a 1-junction:

$$
\begin{array}{ccc}
\overset{\ddot{m_3}}{I_3} & & \overset{\ddot{R_3}}{R_d} \\
\underset{p_I}{\diagdown} & & \underset{p_R}{\diagup} \\
\dfrac{p_\beta}{q_\beta} & \!\!1\!\! & \dfrac{p_\gamma}{q_\beta}
\end{array}
$$

(e) the second chamber #3 has input volume velocity q_β and exit velocity q_γ. Its model is therefore a 0-junction:

$$
\begin{array}{c}
C_3 \\
p_\gamma \Big| q_\beta - q_\gamma \\
\dfrac{p_\gamma}{q_\beta} \;\; 0 \;\; \dfrac{p_\gamma}{q_\gamma}
\end{array}
$$

(f) the second orifice connects chambers #3 and #4. Its graph is,

$$
\begin{array}{ccc}
I & & R \\
\underset{p_I}{\diagdown}{}^{q_\gamma} & {}^{q_\gamma}\underset{p_R}{\diagup} & \\
\dfrac{p_\gamma}{q_\gamma} & \!\!1\!\! & \dfrac{p_\delta}{q_\gamma}
\end{array}
$$

(g) the third chamber #4 has input volume velocity q_γ and exit velocity q_δ (= equalizer-tube velocity), the graph of which is,

$$
\begin{array}{c}
C_4 \\
p_\delta \Big| \\
\dfrac{p_\delta}{q_\gamma} \;\; 0 \;\; \dfrac{p_\delta}{q_\delta}
\end{array}
$$
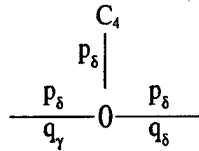
(h) the third "orifice" is the entrance and length the equalizer-tube in which the velocity by approximation is q_δ. Its graph is,

$$
\begin{array}{ccc}
\overset{\ddot{m_f}}{I} & & \overset{\ddot{R_f}}{R} \\
\underset{p_I}{\diagdown}{}^{q_\delta} & {}^{q_\delta}\underset{p_r}{\diagup} & \\
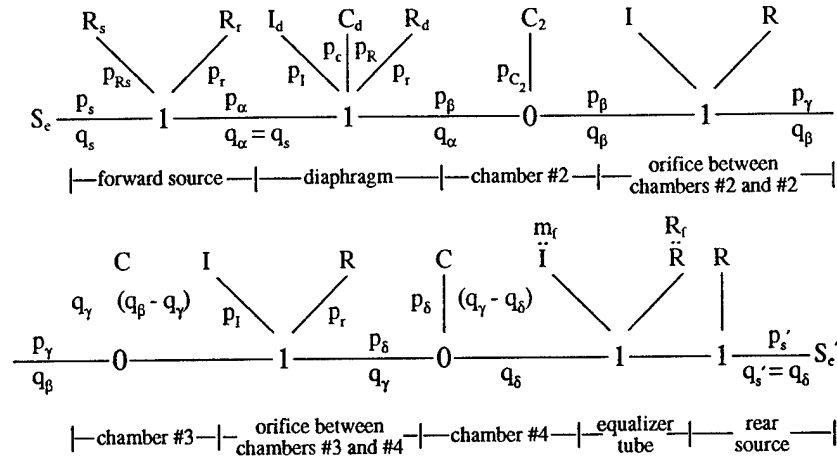\dfrac{p_\delta}{q_\delta} & \!\!1\!\! & \dfrac{p_f}{q_\delta}
\end{array}
$$
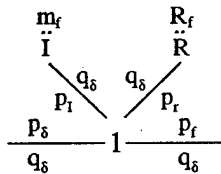
(i) (rear) source
The rear source is the external acoustic pressure impinging on the surface aperture of the equalizer tube. Its graph is <u>structurally</u> the same as the forwarded source but has different magnitudes of the power variables and resistive loads.

Assembly of Components (Cascade Model)

In assembling the components of a bond graph one follows the simple rule that bonds having the same power variables coalesce into a single bond.

Assemblies can be made to various degrees of approximation. We sketch first an assembly which neglects area transformations in the path of flow.

$$
\begin{array}{ccccccc}
R_s & & R_r\ I_d & C_d\ R_d & & C_2 & & I & & R \\
\diagdown & & \diagup\ \diagdown & p_c\ |\ p_R\ \diagup & & | & & \diagdown & & \diagup \\
p_{Rs} & & p_r\ \ p_I & p_r & & p_{C_2} & & p_\beta & & p_\gamma
\end{array}
$$

$$S_e\ \dfrac{p_s}{q_s}\ {-}\ 1\ {-}\ \dfrac{p_\alpha}{q_\alpha = q_s}\ {-}\ 1\ {-}\ \dfrac{p_\beta}{q_\alpha}\ {-}\ 0\ {-}\ \dfrac{p_\beta}{q_\beta}\ {-}\ 1\ {-}\ \dfrac{p_\gamma}{q_\beta}$$

|—forward source —|——diaphragm——|—chamber #2—|— orifice between chambers #2 and #2 —|

$$
\begin{array}{ccccccc}
C & I & R & C & \overset{m_f}{\overset{\cdots}{I}} & \overset{R_f}{\overset{\cdots}{R}} & R \\
q_\gamma\ (q_\beta - q_\gamma) & p_I\ \diagdown & \diagup\ p_r & p_\delta\ |\ (q_\gamma - q_\delta) & \diagdown & \diagup\ R & |
\end{array}
$$

$$\dfrac{p_\gamma}{q_\beta}\ {-}\ 0\ {-}\ 1\ {-}\ \dfrac{p_\delta}{q_\gamma}\ {-}\ 0\ {-}\ \dfrac{}{q_\delta}\ {-}\ 1\ {-}\ 1\ {-}\dfrac{p_s'}{q_s'=q_\delta}\ S_e'$$

|—chamber #3—|— orifice between chambers #3 and #4 —|—chamber #4—|—equalizer tube—|— rear source —|

In a second approximation area transformation can be modeled by use of simple numerical ratio transformers.

Electrical Output

Electrical output of the microphone is accomplished by Faraday induction. In this process the axial velocity V_α of the diaphragm is imparted to the "spool" of wire so that the wire strands move normally to the (d.c.) magnetic flux B which is generated radially by a permanent magnet. By induction an e.m.f. appears across the terminal of the spool of wire, length 1, of amount,

$$E_{o.c} = (\vec{V} \times \vec{B})l$$

For wire resistance R_E the wire electrical current is $i_E = E_{oc}/R_E$. Since E_o is operationally an "effort" it has the character of a negative source (i.e. power output rather than power input). Its bond graph is then

$$
\begin{array}{c}
R_E \\
E_R\ | \\
E_s : S_e\ \dfrac{E_s}{i}\ {-}\ 1\ {-}\ \dfrac{E_\alpha}{i}\ {-}\ \overset{B l v}{\overset{\cdots}{GY}}\ {-}\ \dfrac{F_E}{V_\alpha}\ ;\quad V_\alpha = \dfrac{q_\alpha}{S_\alpha}
\end{array}
$$

The port F_E, V_α is bonded to the 1-junction labeled q_α.

Conclusion

The "plunger-spool" microphone has been modeled above as a lumped parameter device with all components in cascade. In practice the diaphragm will vibrate in multimode patterns of surface displacement. These can be modeled as a "fan of resonant modes" described in Ref. 2.

<u>Bulletin #3 - Transducers, General Principles</u>

In bond graph theory transducers are modeled as multiport systems in which the power variables of bonds are interrelated by coefficients arranged in matrices. For a system of n ports, in which the power variables of the I'th port are ei, fi (effort, flow), the interrelations are expressed by these matrix equations:

(a) to first (or linear) order in ej, fj:

$$e_i = \sum_{j=1}^{n} A_{ij} f_j + \sum_{j \neq i_n}^{n} B_{ij} e_j, \quad i = 1,2,\ldots n$$

$$f_i = \sum_{j \neq i_n}^{n} C_{ij} f_j + \sum_{j=i}^{n} D_{ij} e_j, \quad i = 1,2,\ldots n$$

(b) to second (nonlinear) order in ej, fj:

$$e_i = \sum_{j}^{n} \sum_{k}^{n} A_{ijk} f_j f_k + \sum_{j \neq i_n}^{n} \sum_{k} B_{ijk} e_j e_k + \sum_{j} \sum_{k} E_{ijk} f_j e_k$$

$$f_i = \sum_{j}^{n} \sum_{k}^{n} C_{ijk} f_j f_k + \sum_{j}^{n} \sum_{k}^{n} D_{ijk} e_j e_k + \sum_{j}^{n} \sum_{k}^{n} F_{ijk} f_j e_k$$

Here, the quantities ei, fi on the *l.h.s.* are regarded as dependent variables, while ej, fr on the r.h.s. are taken to be independent variables. The rule for the selection of dependent variables is this:

(a) if an independent power variable is fj, the selected dependent variable is ej

(b) if an independent power variable is ej, the selected dependent variable is fj.

The ei, fi equations noted above are the <u>canonical equations</u> of classic theory. In drawing multiport transducers, the symbol TD is taken to be the bond graph structure of interlinked junctions, i.e. the <u>interior</u> system.

<u>Bulletin #4 - The Electrostatic Transducer</u> (A 2-Port Model)
(Ref.: F. Hunt, "Electroacoustics")

The simplest embodiment of the electrostatic transducer is a parallel plate capacitor having a fixed separation d, a fixed charge qo, a fixed voltage Eo, and an area S. When a squeezing force is applied fluctuating in magnitude and time, a flucutating charge q is induced by the changing separation distance $\dot{x}$, thus generating a fluctuating voltage Ee. The interaction between mechanical and electrical variables is found to be nonlinear.

<u>The Elementary Bondgraph</u>

The electrostatic transducer under time varying input is modeled by transduction T with power variables F_e, $\dot{x}$, and E_e, $\dot{q}$:

$$\frac{F_e}{\dot{x}} \quad T \quad \frac{E_e}{\dot{q}} \; ; \quad F_e = F_e(\dot{q}) \; ; \quad E_e = E_e(\dot{q}\dot{x})$$

Symbol F_e is the force (units: Newtons) between the plates resulting from fluctuating plate charge $\dot{q}$. Symbol E_e is the electrical potential (units: Volts) resulting from the combined effects of fluctuating charge $\dot{q}$ and fluctuating distance $\dot{x}$ between the plates. The nonlinear functional relations between F_e and $\dot{q}$, and between F_e and $\dot{q}$ $\dot{x}$ are:

$$F_e = -\frac{1}{2}\frac{[\int \dot{q}dt]^2}{\varepsilon_o S} ; \quad E_e = \frac{\int\int \dot{q}\dot{x}dtdt}{\varepsilon_o S}$$

Symbol ε_o is the permittivity of free space (units: coulombs/voltmeter). From these choices of units, transduction T is derived to have the units: Volt meter/Newton Coulomb.

A simple acausal bond graph of a generalized electrostatic transducer may now be sketched:
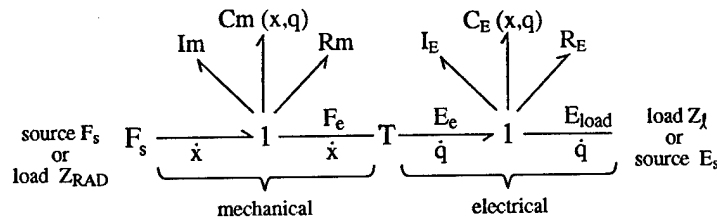
This graph may be read as follows: an applied exterior source force F_s overcomes the reactive mechanical impedance of the transducer (namely I_m, C_m, R_m) leaving a residual force F_e due to the attraction (or repulsion) between the plates resulting from the effects of fluctuating charge. The direction of the bond of F_e is left for further discussion. The transduced voltage E_e, $\dot{q}$ to satisfy a reactive load Z_l. The transduction symbol T is bonded to two 1-junctions, hence it simulates a 0-junction, i.e., it acts as a "coupler."

Biased Operation

In practice electrostatic transducers operate with permanent electrostatic bias, furnished in the simplest structures, by an electric battery. To illustrate biasing, we choose the case of an elementary capacitor mode of a flexible diaphragm, area S, electroded on its inner surface, set at a distance d from a perforated metal screen which is at ground potential. When a d.c. bias voltage is applied to the capacitor in the form of E_o (volts), a charge q_0 will appear on the plates, and simultaneously a steady diaphragm displacement xo from equilibrium will occur. For a given mechanical compliance Cm of the diaphragm there will be an effective mechanical force X_o/C_m tending to restore the diaphragm to its initial shape. In the absence of applied (= external) mechanical forces, this restoration force will be balanced by the electrically generated force of attraction,

$$0 = \frac{x_o}{C_m} + \frac{q_o^2}{2\varepsilon_o S} ; \quad V_0 = \frac{q_o(d+x_o)}{\varepsilon_o S}; \quad C_0 \doteq \frac{\varepsilon_o S}{d+x_o}$$

These equations lead to an implicit equation for x_0 of third degree,

$$-X_0 = C_m V_0^2 \frac{\varepsilon_o S}{2(d+x_o)^2}$$

Values of x_0 can be obtained by graphical procedures.

Two modes of biased operation are used in practice

(a) transmitter mode, in which an electric source generates, through transduction, an acoustic field.

(b) receiver mode, in which an acoustic field, acting as a mechanical source generates an electrical signal.

Restricting attention to the receiver mode, it is assumed that a fluctuating acoustic pressure acting over area S of the diaphragm is equivalent to a concentrated fluctuating force F_1. Such a force generates a time fluctuating displacement X_1, together with a time fluctuating charge q_1. At any time then, total displacement X and total charge q are

$$X = d + X_0 + X_1; \quad q = q_0 + q_1$$

Expanding q^2 and qx, and neglecting second order terms, one finds the first order terms in fluctuating q_1, x_1 to be $2 q_0 q_1$ and $q_0 x_1$. The forces and voltages of electromechanical coupling, to first order, namely $F_{me(1)}$ [read, mechanical force due to a fluctuating q1] and $E_{em(1)}$ [read, electrical voltage due to fluctuating x_1] are

$$F_{me(1)} = \frac{q_0 q_1}{\varepsilon_0 S} = \frac{-I_1}{jw\, C_{em}}; \quad V_{em(1)} = \frac{q_0 X_1}{\varepsilon_0 S} = \frac{v_1}{jw\, C_{em}}; \quad C_{em} = \frac{\varepsilon_0 S}{q_0}$$

The units of C_{RM} are m/V or C/N (meters/volt or coulomb/Newton). These expressions allow one to form acausal equations by use of the bond graph. At the 1-junction of the mechanical side of summation of forces on the diaphragm leads to $F_1 = Z_m v_1 \pm F_{me(1)}$. The ambiguity in the sign of Fme(1) is resolved by noting that the time fluctuating charge q1 acts as a <u>source</u> so that the force balance is $F_1 + F_{me(1)} = Z_m v_1$, from which

$$F_1 = \frac{q_0 q_1}{\varepsilon_0 S} + Z_m \dot{X}_1 = \frac{-I_1}{j\omega\, C_m} + Z_m v_1 = T_{me(1)} + Z_m v_1$$

Here, T_{me} (which is $1/j\omega\, C_{em}$) is the transduction coefficient. This choice of sign is justified on physical grounds. When F_1 increases, closing the gap, the electrostatic force of attraction also increases; similarly, when F_1 decreases, opening the gap (between diaphragm and screen), the electrostatic force decreases. Thus, the fluctuation of charge appears on the mechanical side as a force source adding to the applied force F_1. On the electrical side of transduction the electrical effect of gap motion in the form of displacement X_1 is $E_{em(1)} = q_0 X_1/\varepsilon_0 S = X_1/C_{em}$. This is the potential generated by the fluctuation X_1. It's direction is <u>away</u> from the 1-junction labelled $\dot{q}$, that is, it constitutes a negative source. In absence of an applied electric potential at the electrical terminals, summation at the 1-junction $\dot{q}$ leads to,

$$Z \dot{q}_1 = V_{me(1)}, \quad Z = Z_E + Z_\ell, \quad Z_\ell = \text{electrical load}$$

In terms of first order fluctuating variables, this summation of potentials is,

$$O = Z \dot{q}_1 + \frac{X_1}{C_{em}}$$

Assume now that fluctuation in time is sinusoidal. By elimination of $\dot{q}_1$ one finds

$$F_1 = \dot{X}_1 \left[Z_m - \frac{1}{(j\omega\, C_{em})^2 Z} \right]$$

In absence of an electrical load, at low frequencies,

$$Z_m \underset{\omega \to 0}{=} \frac{1}{j\omega\, C_m}; \quad Z_E \underset{\omega \to 0}{=} \frac{1}{j\omega\, C_o};$$

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Thus as $\omega \to 0$,

$$F_1 \underset{\omega \to 0}{=} \frac{X_1}{c_m}\left[1 - \frac{c_m C_o}{C_{em}^2}\right] = \frac{X_1}{C_m'}$$

Here the quantity $c_m C_c / C_{cm}^2$ describes the coupling between mechanical and electrical 'compliances' at very low frequency. It is designated as k^2, the coefficient of electromechanical coupling (squared),

$$C_m^{-1} = \frac{C_m}{1-k^2}; \; K^2 = \frac{c_m C_o}{C_{em}^2}; \; C_0 = \frac{\varepsilon_0 S}{dt\, xo}; \; C_{em} = \frac{\varepsilon_0 S}{q_0} = \frac{dt\, X_0}{E_0}$$

These formulas show k^2 to be a function of E_0, q_0, X_0, and of diaphragm compliance C_m. From energy considerations there is a limit on the magnitude of k^2,

$$k^2 \geq 1$$

This is also a requirement for stability of the transducer. For given C_m the achievable bias displacement X_0 is limited,

$$\frac{1}{C_m} \geq \frac{E_0^2 \varepsilon_0 S}{(dt\, x_0)^3}$$

It can also be deduced from these formulas that the low frequency electrical capacitance C_s', also called the 'free-running capacitance' is

$$C_0' = \frac{C_o}{1-k^2}, \text{ where } C_o \text{ is the 'blocked capacitance.'}$$

Equivalent Circuit; Ideal Transformer; Turns Ratio

Although every equivalent circuit may be transcribed into a bond graph, a bond graph is not, per se, an equivalent circuit. Steps required to form a 2-port equivalent circuit of an electrostatic transducer are outlined next.

(1) Let E_1, F_1 be the terminal electric potential and mechanical force respectively of the 2-port. On the electrical side the blocked electrical capacitance C_0 is moved to 'shunt' position, that is, from a 1-junction to a 0-junction:
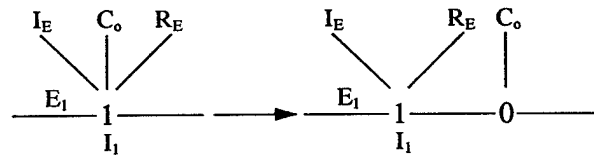
The purpose of this move is to make C_0 a 'coupler' between I_1 and v_1.

(2) A ratio ϕ is formed of the transduction coefficient T_{em} and $Z_E = 1/j\omega\, C_0$,

$$\phi = \frac{1/j\omega\, C_{em}}{1/j\omega\, C_0} = \frac{C_o}{C_{em}} \; \left(\text{units: } \frac{C}{m} \text{ or } \frac{N}{V}\right)$$

This quantity is the "terms ratio" of an ideal electromechanical transformer which relates the independent variables I_1, v, of the canonical equations to the dependent variables F_1, E_1:
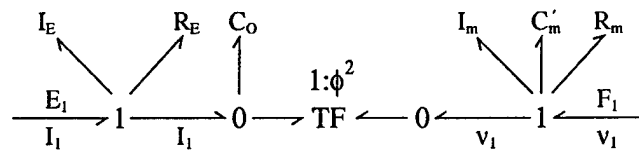
$$E_1 \phi = F_1; \quad I_1 = \phi v_1; \quad E_1/I_1 = F_1/v, \phi^2, \text{ i.e. } Z_E = Z_m/\phi_2$$

In terms of the bias quantities E_0, X_0, q_0,

$$\phi = \frac{E_o C_o}{dt \, x_o} = \frac{q_o}{dt \, x_o}$$

By shifting C_0 to the shunt position, the mechanical compliance changes from cm to C_m', a result of making the blocked capacitance C_0 the coupling element.

The 2-port equivalent circuit is now complete: its bond graph is,

The arrow directions are conventional for a 2-port electromechanical quivalent circuit.

<u>Bulletin #5 - Mathematical Modeling of Electromechanical Transducers</u>
<u>Based on Electrical Four-Pole Theory</u>
(Ref.: "Grundlagen Der Technischen Akustik," W. Reichhardt, Leipzig, 1968)

In electrical 4-pole theory the power variables are E_1, i_1, E_2, i_2. Chosen as independent variables are E_2, i_2, which implies the functional relations,

$$E_1 = \varphi_1 (E_2, i_2)$$

$$i_1 = \varphi_2 (E_2, i_2)$$

To a first linear approximation, one finds,

$$E_1 = A_{11} i_2 + A_{12} E_2 \quad \text{or} \quad \begin{bmatrix} E_1 \\ i_1 \end{bmatrix} = A \begin{bmatrix} i_2 \\ E_2 \end{bmatrix}; \quad A = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}$$
$$i_1 = A_{21} i_2 + A_{22} E_2$$

In general all power variables are complex phasors in steady state operation, requiring (then) all coefficients Aij to be complex numbers.

There are limits to the components Aij. These are set by the assumption that the transfer of power through the 4-pole is lossless. It is then seen that,

$$\tfrac{1}{2} (E_1 i_1^* + E_1^* i_1) - \tfrac{1}{2} (E_2^* i_2 + E_2^* i_2) = 0$$

From this requirement one deduces that,

(a) $(A_{11} A_{21}^* + A_{11}^* A_{21}) \, i_2 \, i_2^* = 0$

(b) $(A_{12} A_{22}^* + A_{12}^* A_{22}) \, E_2 \, E_2^* = 0$

(c) $(A_{11} A_{22}^* + A_{12}^* A_{21} - 1) \, i_2 \, E_2^* = 0$

(d) $(A_{12} A_{21}^* + A_{11}^* A_{22} - 1) \, i_2^* E_2 = 0$

Two types of transducers can be modeled by the E1, i1, equations noted above:

(1) E1 $\propto$ i2 for the (short circuit condition) $E_2 \equiv 0$. In from (a) above A_{21}, is purely imaginary (written $j A_{21}^{\perp\perp}$). A_{22} (at our option) is made purely real which makes A_{12} purely imaginary (written $A_{12}^{\perp\perp}$), as seen from (b). These assignments are inserted into (c) resulting in

$$A_{11}^{\perp} A_{22}^{\perp} - j A_{12}^{\perp\perp} j A_{21}^{\perp\perp} = |A| = +1 \, ,$$

(2) In the second type $E_1 \propto E_2$ for the (open circuited) conduct $i_2 \equiv 0$. In this case A_{12} is purely real (written $A_{21}^{\perp}$) because E_1 is assumed purely real. From (b) above it is seen that A_{22} is purely imaginary (written $A_{22}^{\perp\perp}$). A_{21} (at our option) is made purely real, which from (a) above makes A_{11} purely imaginary (written j $A_{21}^{\perp\perp}$). These assignments inserted in (d) result in the formula

$$A_{12}^{\perp} A_{21}^{\perp} - j A_{11}^{\perp\perp} j A_{22}^{\perp\perp} - 1 = 0$$

or

$$j A_{11}^{\perp\perp} j A_{22}^{\perp\perp} - A_{12}^{\perp} A_{21}^{\perp} = |A| = -1$$

This expression for the determinant of A, as well as the one in (1) above can serve to characterize electromechanical transducers.

Four-Pole Modeling of Electromechanical Transduction

The transduction component of a transducer can be modeled as a 4-pole. In bond graph rotation, the power variables Em, v, Fe appear as bonds,

$$\frac{E_m}{i} \, T \, \frac{v}{F_e}$$

E_m = electric field coupled to the mechanical circuit
F_e = mechanical force coupled to the electrical circuit
i = current
v = mechanical velocity

Relations between variables are based on v, Fe, chosen as independent variables, and Em, i, as dependent variables.

$$E_m = A_{11} \, v + A_{12} \, F_e$$

$$i = A_{21} \, v + A_{22} \, F_e$$

Two types of transduction may be deduced.

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(1) In the transfer term A_{12} v one sets v ≡ 0. Then Em = A_{11} Fe. The determinant $|A| = -1$. This type of transduction characterizes the magnetic field transducer

(2) In the transfer term A_{21} v one sets v ≡ 0. Then Em = A_{12} Fe. The determinant $|A| = -1$. This type of transduction characterizes the electric field transducer.

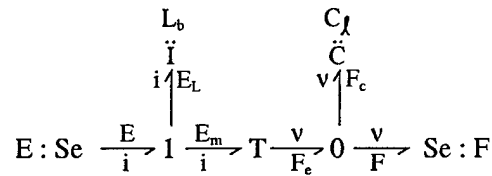The transduction component is one link in a chain (= cascade) of transducer components, every link of which is itself a 4-pole.

Four-Pole Low Frequency Canonical Equations and Corresponding Bond Graphs

Magnetic Field Transducers

A low frequency bond graph has the general structure given by

$$— 1 — T — 0 — \quad (= \text{"one-tee-zero"})$$

In detail,

$$\begin{array}{ccc}
L_b & & C_f \\
\ddot{I} & & \ddot{C} \\
i\uparrow E_L & & v\uparrow F_c \\
\end{array}$$

$$E : Se \xrightarrow[i]{E} 1 \xrightarrow[i]{E_m} T \xrightarrow[F_e]{v} 0 \xrightarrow[F]{v} Se : F$$

(Arrows show transmission from electrical field to mechanical field.)

Explanation

(1) The source Se at the left is a voltage (= effort) source and must therefore feed into a 1-junction.

(2) The source Sf at the right is a force (= flow) source and must therefore be bonded to a 0-junction. This is the result of representing the mechanical side by its dual form.

(3) The blocked coil inductance Lb is bonded to the 1-junction (an effort divider) to represent the physical requirement that for the condition $\omega_c \to 0$ the voltage $E_L \to 0$, leaving source E to be equal to Em. If Lf was attached to a 0-junction the source would be short-circuited and no d.c. current could flow.

(4) The mechanical compliance C_f is bonded to the common velocity (dual form) 0-junction to represent the physical requirement that for the condition $\omega \to 0$ the force F_c becomes infinite (= "open circuit") and the velocity (i.e. the a.c. component) becomes zero (= clamped condition). Bonding C_f to a 1-junction (in dual form) would mean $v_c \to 0$ as $\omega \to 0$ so that $v_e \to n$ leading to a finite velocity at zero frequency, which is unphysical.

(5) Writing for brevity the transduction coefficient as X, one may codify the transduction equations as,

$$\begin{array}{l} i = X\, F_e \\ E_m = \dfrac{1}{X} v \end{array} \quad ; \quad (\text{units of } X : \frac{C}{sN} ; \frac{1}{X} : \frac{Vs}{m})$$

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The bond graph equations are found by summing voltage at the 1-junction and force at the 0-juntion,

$$E = jw\, L_G\, i + \frac{1}{X}\, v$$

$$i = X\left[\frac{v}{jw\, Cn} + F\right]$$

Substitution of i into the E equation leads to the canonical set in v, F, whose matrix is,

$$A = \begin{pmatrix} A_{11}^{\perp} & jA_{12}^{\perp\perp} \\ jA_{21}^{\perp\perp} & A_{22}^{\perp} \end{pmatrix}; \quad A_{11}^{\perp} = \frac{L_G}{Cn}\, X + \frac{1}{X}\, ; \; jA_{12}^{\perp\perp} = j\, WL_G\, X$$

$$jA_{21}^{\perp\perp} = \frac{X}{jW\, Cn}; \; A_{22}^{\perp} = X$$

The determinant $|A| = +1$, as required.

Electric Field Transducers

A low frequency bond graph has the general structure given by,

$$— 0 — T — 0 — \text{(= "zero-tee-zero")}$$

In detail,
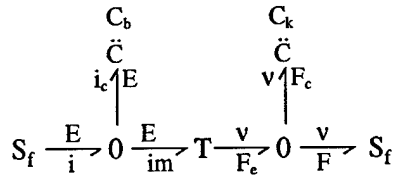
It is seen that the electrical field is in "direct form" and the mechanical field is in "dual form." The choice of dual form comes from the choice of v, F as independent variables.

Explanation

(1) The blocked capacity C_b is bonded to a 0-junction to represent the physical requirement that for the condition $\omega \rightarrow 0$ the current i_c vanishes and the "supply current" i becomes the current ** coupled to the mechanical circuit.

(2) The source driving the blocked capacity must be a 'current source' S_f. Ordinarily, this type of source is furnished by a 'voltage source' S_e bonded to a 1-junction which is loaded with a source impedance Z_s:

(3) Writing for brevity the transduction coefficient as Y, one may codify the transduction equations as,

$$E = Y\, F_c$$
$$i_m = \frac{1}{y}\, v \quad ; \quad (\text{units of } Y: \frac{V}{N};\ \frac{1}{Y}: \frac{m}{C})$$

The bond graph equations are found by summing currents at the 0-junction and forces at the (dual) 0-juntion:

$$i = jw\, C_b\, E + \frac{1}{y}\, v$$
$$E = Y\, Fe = Y\left[\frac{v}{jw\, Cm} + F\right]$$

Substitution of E into the i equation leads to the canonical set in v, F, whose matrix is,

$$A = \begin{pmatrix} jA_{11}^{\parallel} & A_{12}^{\perp} \\ A_{21}^{\perp} & jA_{22}^{\parallel} \end{pmatrix} ; \qquad jA_{11}^{\parallel} = \frac{Y}{jW\, Cn};\ A_{22}^{\perp} = Y$$

$$A_{21}^{\perp} = \frac{C_b}{Cn}\, Y + \frac{1}{Y};\ jA_{22}^{\parallel} = jW\, C_b\, Y$$

The determinant $|A| = -1$, as required.

Conclusions

Modeling electromechanical transducers on the basis of electrical 4-pole theory involving the power variables E, i, v, F, leads to a set of canonical equations in which choices of independent variables must be made. For various reasons several authors choose v and F, with the additional choice of 'v across' and 'F through.' The mechanical branch is then represented in <u>dual</u> form. Other authors take i,v to be independent variables, both in <u>direct</u> form, namely 'i through', 'v through.' The resulting canonical set, while different from the set based on 4-pole theory, can be converted to it by interchange of variables.

Basic transducer types are simply summarized:

Magnetic field; — 1 — T — 0—; driven by a "voltage source"
Electric field; — 0 — T — 0 —; driven by a "current source"

Coefficient of Electromechanical Coupling and Its Effect on Low Frequency Properties of Inductance, Capacitance and Compliance. Case I. F, V at right angles to E, D or B, H.

The bond graph that describes the low frequency behavior of a magnetic field transducer calls for a blocked inductance Lb bonded to a 1-junction and a mechanical compliance bonded to a 0-junction (dual form). Because of electromechanical coupling the magnitude of the compliance depends on the nature of the electrical drive. Since magnetic field transducers are driven by a voltage source a purely (uncoupled) mechanical magnitude of compliance can be measured by opening the electric input circuit. The open circuit compliance is then written a C_l. When the circuit is closed and a current i flows through the (coil) inductance L_G the open circuit compliance transferred to the electrical branch appears as coupled inductance $L_c = C_l/X^2$ bonded to the 1-junction. The sum $L_b + L_c = L_f$, where L_f is the "free-running" inductance. By defining the coupling coefficient $K^2 = L_c/(L_c + L_b)$, it is readily deduced that the measured free-running inductance $L_f = L_b/1-k^2$. Further more it may readily be demonstrated that the measured compliance C_k on short circuit drive is related to the open circuited compliance C_l by the formula, $C_k = C_l(1-k^2)$. It also may readily be seen that because $k^2 \le 1$ the short circuit compliance is less than the open circuit compliance.
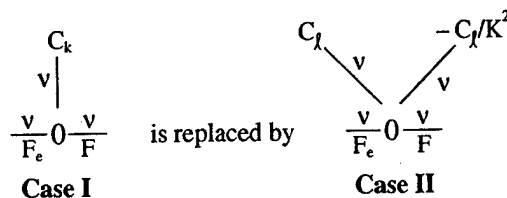
20

Similar considerations can be applied to electric field transducers. Since they are driven by a current source the mechanical compliance C_k is measured at electrical short circuit thereby bypassing the blocked capacity C_b. When a drive voltage is restored the mechanical compliance <u>decreases</u> to the "free-running" magnitude $C_l = C_k (1-k^2)$. Correspondingly, the "free-running" electrical capacity C_f is <u>increased</u> to $C_f = Cb/1-k^2$.

<center>Case II. F, v parallel to E, D or B, H</center>

In this case negative compliances appear (see Bulletin on the electrostatic transducer). Summarizing the changes to be made to Case I:
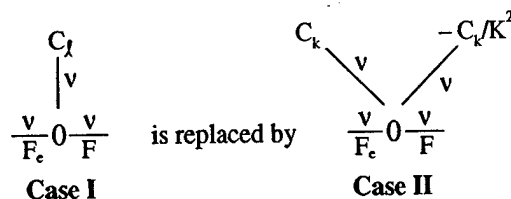
Electric field transducers

short circuit compliance C_k is replaced by open circuit compliance C_l plus a negative compliance $-C_l/k^2$; i.e.

<center>
C_k

$\dfrac{v}{F_e} 0 \dfrac{v}{F}$ is replaced by $C_l \quad -C_l/K^2$ $\dfrac{v}{F_e} 0 \dfrac{v}{F}$

Case I   **Case II**
</center>

Magnetic Field Transducers

open circuit compliance C_l is replaced by short circuit compliance N_k plus a negative compliance $-C_k/k^2$; i.e.

<center>
C_l

$\dfrac{v}{F_e} 0 \dfrac{v}{F}$ is replaced by $C_k \quad -C_k/K^2$ $\dfrac{v}{F_e} 0 \dfrac{v}{F}$

Case I   **Case II**
</center>

The rationale for these changes is explained by,

A. Lenk, Acoustica (1958) Nr. 3, page 159

<center>

<u>Bulletin #6 - Moving Armature Transducer</u>
(Ref.: "Electric Circuits," MIT, J. Wiley & Sons, Inc., New York, 1943)
</center>

The Moving Armature Transducer (shown in the simplified sketch below) is based in operation on energy transfer between electrical and mechanical ports associated with time variation of reluctance in a magnetic circuit. As shown it has three ports: (1) a field winding of electrical wire, N_f turns, around an iron yoke. Its power variables are potential difference v_f (units: volt) and current i_f (units: amp.) (2) an armature winding of electrical wire, N turns, around the pivoted iron armature.
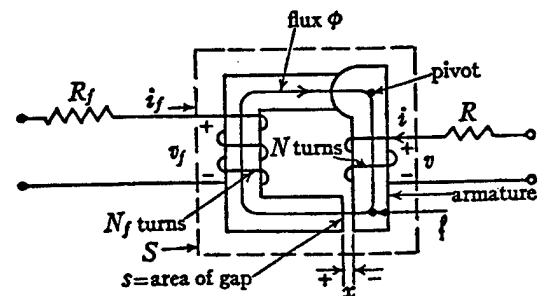
Moving-iron mechanism.

<center>21</center>

Its power variables are v (units: volt) and i (units: amp). (3) a mechanical port near the gap of the magnetic circuit. Its power variables are force F (units: newton) and velocity V (units: meter/sec).

In operation the gap energy Wm for a magnetic feed strength H (units: amp/meter) and magnetic flux density B (units: Vs/m^2) is $(HB/8\pi)$ Sx = W_m, where Sx is the gap volume. When a force F is applied at the mechanical port, it changes the gap with x. The gap magnetic energy changes in the amount $dW_m/dx = F$. Allowing H = $B/\mu o$ in the gap it is seen that $F = SB^2/8p\ \mu o$.

At the electrical port of the field winding energy can only be extracted in the amount $i_f\ d\ (N_f\varphi)/dt$. The field current i_f is nearly constant and the flux φ (units: Vs), varying around a constant mean value, is also nearly independent of time to the approximation used in this analysis. Thus over time the field energy W_f at the field port does not change.

At the electrical port of the armature winding the rate of energy extraction when armature current i is active is $dWa/dt = Nid\varphi/dt$. Since current i is variable due to variable F, and flux φ in the gap varies with changing gap δx, then energy may be transferred at the armature winding. The total flux in the gap is the sum of a constant component due to $N_f\ i_f$ and a variable component Ni

$$\varphi = \frac{4\pi\ (N_fi_f + N_i)\ S\mu_o}{(X_o + \Delta X)} = \frac{4\pi\ S\mu_0\ N_fi_f\ (1 + \frac{N_i}{N_f})}{X_o\ (1 + \Delta X/X_o)}$$

If Ni/Ni_f and $\delta x/Xo$ are small relative to their squares, when the division is made and second order terms rejected the flux in the gap is approximated by

$$\varphi \simeq \frac{4\pi\ S\mu_o\ N_fi_f}{X_o}\ (1 + \frac{N_i}{N_fi_f} - \frac{\Delta X}{X_o})$$

Since this flux varies with time and displacement there is a voltage v induced in the armature winding,

$$v = N\frac{d\varphi}{dt} = L_b\frac{di}{dt} = B_{f_o} = \frac{SN}{X_o}\ v\quad (v = velocity)$$

where

$$L_b = \frac{4\pi\ S\mu_o\ N^2}{X_o}\ ;\quad B_{f_o} = \frac{4\pi\ \mu_o\ N_fi_f}{X_o}\quad (units:\ Vs/m^2)$$

It is seen that L_b is the self-inductance of the armature winding. Thus the induced armature voltage has two components: the first in di/dt is the voltage of self induction, and the second an e.m.f. generated by mechanical motion.

It was noted earlier that the applied force in the magnetic circuit air gap is $F = S\ B^2/8\pi\ \mu o$, where B is the flux density. Noting that B = φ/S and rejecting $(Ni/N_fi_f)^2$, $(\Delta x/xo)^2$, $(Ni/N_fi_f)\ (\Delta x/xo)$, as terms of second order, it is seen that to first order,

$$F = \frac{2\pi\ S\mu_o\ (N_fi_f)^2}{X_o^2}\ (1 - \frac{2\pi\ \Delta x}{X_o} + \frac{2Ni}{N_fi_f}\ ...\ )$$

Each term in this 3-term form has significance. (1) the first term states that there is a component of force due to current i_f of fixed magnitude tending to close the gap. (2) the second term states that there is a negative component of force proportional to Δx which tends to increase the displacement, that is the application of this force <u>widens</u> the gap. (3) the third term states that there is a component of force that depends on armature current i, that is, when time varying current is is impressed in the armature ciruit it causes the armature to move in unison with it, and when the armature is moved by a mechanical force an e.m.f. in the electrical port is generated as noted above.

<u>Construction of the Bond Graph</u> (Ref.: Reichhardt, see above).

Since the field current i_f is virtually constant and the armature current i is variable with time some analysts combine their respective ports into one electrical port, the power variables of which are u (units: volt) and i_{ges} (units: amp) where $i_{ges} = i_f + i$. We adopt this procedure here. Now the magnetic flux due to i_{Fe} is $N^2 l / R_{Fe}$, where R_{Fe} is the reluctance of the magnetic circuit (units: C/VS^2). The input admittance of the combined electrical port is $R_{Fe}/j\,WN_f^2$ where N_f is the total turns of the field wire (units: none). The first element of the bond graph is a current divider (i.e. a 0-junction)

$$
\begin{array}{c}
L_{Fe} \\
\big| \, i_{Fe} \\
\underline{\phantom{i_{ges}}}\, 0 \, \underline{} \qquad i_{Fe} = \dfrac{u\,R_{Fe}}{jw\,N_f^2}. \\
i_{ges} \quad u \quad i
\end{array}
$$

The second element of the graph involves the armature current i. At the gap the magnetic flux $\Phi gap = Ni\mu_0 S / xo$ (units: VS). The drop in voltage across the gap is $j\omega\,N\Phi gap$ or $j\omega\,L_b i$, $L_b = N^2\,\mu_0 S/Xo$. This is the input impedance of the current i. It's graph is a 1-junction:

$$
\begin{array}{c}
L_b \\
\big| \, (u-u_w) \\
\underline{}\, 1 \, \overset{u_w}{\underline{}} \qquad u-u_w = jw\,L_b\,i \\
i
\end{array}
$$

The third element in the graph is the transduction. In electromagnetic circuits it is a gyrator, defined as the 'gyration' of electrical voltage (u) into mechanical velocity v (units: m/s), and the 'gyration' of electrical current into mechanical force F (units: Newton). Its graph is

$$
\overset{X^*}{\underset{F_w}{\underline{u_w}\, Gy\, \underline{}}} \qquad u_w = X^* v \,; \qquad F_w = X^* i
$$

Here, U_N is an 'across' variable and F_w is a 'flow' variable.

From earlier discussion,

$$
X^* = B_f\,\frac{SN}{X} \qquad \text{where} \qquad B_f = \frac{4\pi\,N_f\,i_f\,\mu_0}{X} \qquad \left(\text{units: } \frac{V_s}{m^2}\right)
$$

and, X has the units of Vs/m.

The fourth element in the graph is the mechanical circuit. In this circuit is armature described dynamically by lumped parameters of mass M stiffness K mechanical resistance R_M and radiation resistance R_{RAD}. These are all attached to a flow (i.e. force) divider, which is a 0-junction:
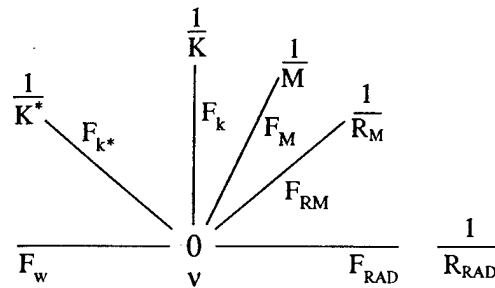
Here, F_{K*} is the force of the negative stiffness discussed earlier.

The Completed Graph
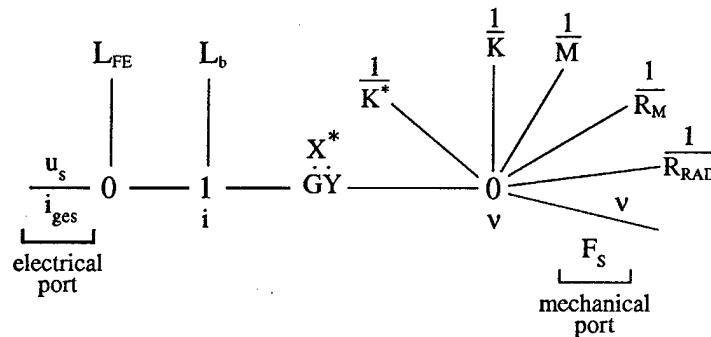
Application to the Telephone

In the telephone, the human voice supplies a variable pressure to a metal disphragm. This is the armature. It vibrates with a velocity which is transduced into an electrical current, which is transmitted to a distant receiver. At the receiver a voltage is generated upon reception of current. This voltage is transduced into a mechanical velocity of a diaphragm which radiates sound, completely reproducing the message sent by the transmitter.